

Today

19/2 ①

- ① Lagrangian Mechanics
(crash course)
- ② Linearizations
- ③ State space equations
for composite systems.

Modeling w/ Lagrange's eqs.

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For any mechanical system

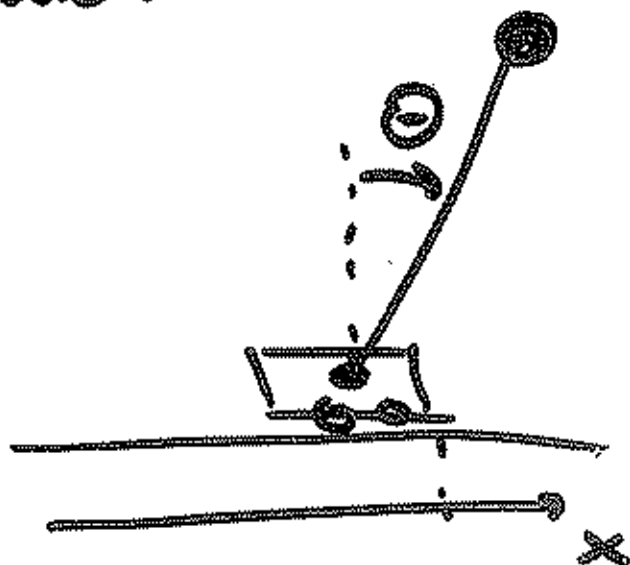
q_i : generalized coordinates

$\dot{q}_i$: generalized velocities

Ex: Inverted pendulum

x : cart position

θ : arm angle



The Lagrangian L is defined by

$$L := T - V$$

$T = \text{kinetic Energy}$

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$T(q, \dot{q})$ function of position & velocities

$V = \text{Potential Energy}$

$V(q)$ function of position

Equation of Motion:

Euler-Lagrange eqs.

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = F_i$$

$i=1, \dots, n$

F_i : generalized forces

account work done by
non-conservative or external
forces

recall that

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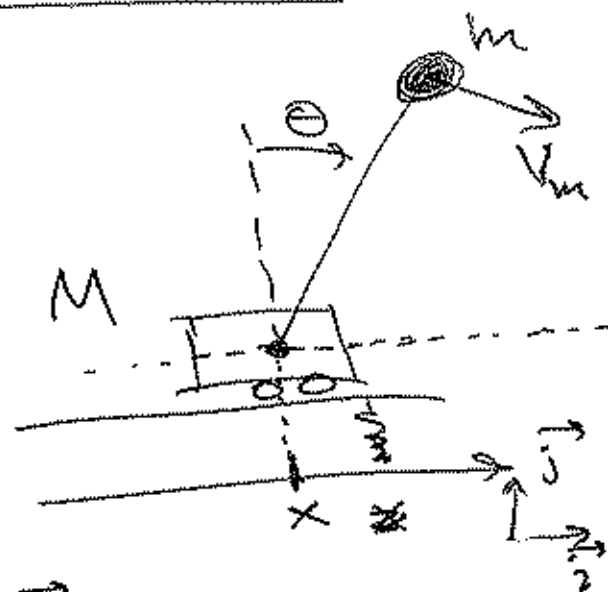
$$L(q_1, \dot{q}_1, \dots, q_n, \dot{q}_n) = T(q_1, \dot{q}_1, \dots, q_n, \dot{q}_n) - V(q_1, \dots, q_n)$$

Pendulum Example:

$$V = \dot{x}$$

$$\omega = \dot{\theta}$$

$$T = \frac{1}{2} M V^2 + \frac{1}{2} m V_m^2$$



$$\vec{V}_m = \frac{d}{dt} (x + l \sin \theta) \vec{i} + \frac{d}{dt} (l \cos \theta) \vec{j}$$

$$= (\dot{x} + l \dot{\theta} \cos \theta) \vec{i} - l \dot{\theta} \sin \theta \vec{j}$$

$$V_m^2 = \dot{x}^2 + l^2 \dot{\theta}^2 \cos^2 \theta + 2 \dot{x} l \dot{\theta} \cos \theta + l^2 \dot{\theta}^2 \sin^2 \theta$$

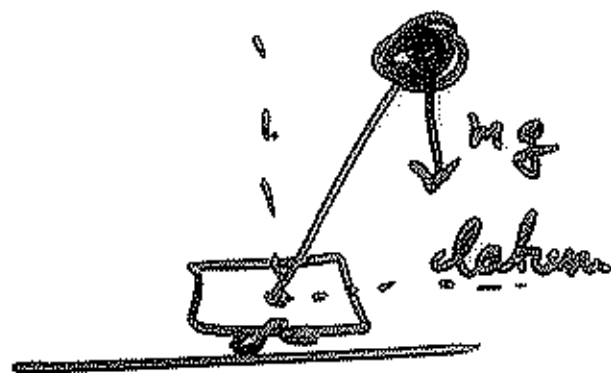
$$= \underline{\dot{x}^2} + l^2 \underline{\dot{\theta}^2} + 2 \underline{\dot{x} l \dot{\theta} \cos \theta}$$

$$T = \frac{1}{2}(M+m) \underline{v^2} + \frac{1}{2} m l^2 \underline{\omega^2} + m \underline{v} l \underline{\omega} \cos \underline{\theta}$$

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Potential Energy

$$V = m g l \cos \theta$$



$$L = T - V$$

$$= \frac{1}{2}(M+m) \underline{v^2} + \frac{1}{2} m l^2 \underline{\omega^2} + m \underline{v} l \underline{\omega} \cos \underline{\theta} - m g l \cos \underline{\theta}$$

$$\frac{\partial L}{\partial q_i}, \frac{\partial L}{\partial \dot{q}_i} =: \frac{\partial L}{\partial x}, \frac{\partial L}{\partial \theta}, \frac{\partial L}{\partial v}, \frac{\partial L}{\partial \omega}$$

$$\frac{\partial L}{\partial x} = 0$$

$$\frac{\partial L}{\partial \theta} = -m v l \omega \sin \theta + m g l \sin \theta$$

$$\frac{\partial L}{\partial v} = (M+m) v + m l \omega \cos \theta$$

$$\frac{\partial L}{\partial \omega} = ml^2 \omega + mvl \cos \theta \quad (6)$$

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \omega} \right) = ml^2 \dot{\omega} + m \dot{v} l \cos \theta - m v l \dot{\theta} \sin \theta$$

$$\frac{d}{dt} \left(\frac{\partial L}{\partial v} \right) = (M+m) \dot{v} + ml \dot{\omega} \cos \theta - ml \omega^2 \sin \theta$$

temporarily
assume external forces are zero

$$\frac{d}{dt} \left(\frac{\partial L}{\partial v} \right) - \frac{\partial L}{\partial x} : (M+m) \dot{v} + ml(\dot{\omega} \cos \theta - \omega^2 \sin \theta)$$

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \omega} \right) - \frac{\partial L}{\partial \theta} : ml^2 \dot{\omega} + m \dot{v} l \cos \theta - m l g$$

in matrix form

$$\begin{bmatrix} M+m & ml \cos \theta \\ \cos \theta & l \end{bmatrix} \begin{bmatrix} \dot{v} \\ \dot{\omega} \end{bmatrix} = \begin{bmatrix} \omega^2 \sin \theta m l \\ g \end{bmatrix}$$

invert
to get

$$\begin{aligned} \dot{v} &= f_1(x, \theta, v, \omega) \\ \dot{\omega} &= f_2(x, \theta, v, \omega) \end{aligned}$$

Observe: The state variables 7
are generalized coordinates
= velocities

$$\text{state: } \begin{bmatrix} x \\ \theta \\ v \\ \omega \end{bmatrix}$$

state equation:

$$\frac{d}{dt} \begin{bmatrix} x \\ \theta \\ v \\ \omega \end{bmatrix} = \begin{bmatrix} v \\ \omega \\ f_1(x, \theta, v, \omega) \\ f_2(x, \theta, v, \omega) \end{bmatrix}$$

Generalized Forces:

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$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = F_i$$

dW_i = total work done by all
non-conservative forces
when q_i changes by dq_i
and all others fixed

$$dW_i = F_i dq_i$$

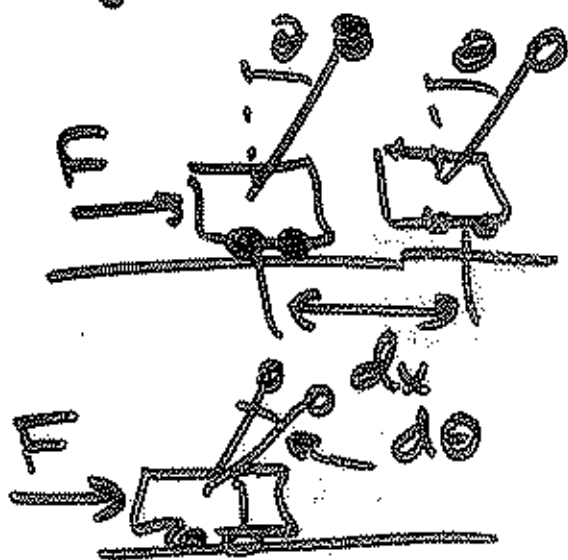
definition
of F_i

$$dW_x = F dx$$

$$\therefore F_x = F$$

$$dW_\theta = 0 = F_\theta d\theta$$

$$\therefore F_\theta = 0$$



Linearization

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General nonlinear state
space system:

$$\dot{x} = f(x)$$

$\Leftrightarrow$

$$\dot{x}_1 = f_1(x_1, \dots, x_n)$$

$\vdots$

$$\dot{x}_n = f_n(x_1, \dots, x_n)$$

if $\bar{x}$ is such that $f(\bar{x}) = 0$

$\Leftrightarrow$

$$f_1(\bar{x}_1, \dots, \bar{x}_n) = 0$$

$\vdots$

$$f_n(\bar{x}_1, \dots, \bar{x}_n) = 0$$

then $\bar{x}$ is called an equilibrium
point

if $\bar{x}$ is s.t. $f(\bar{x}) = 0$ (10)

the consider

$$\dot{x} = f(x)$$

$$\dot{x}(t) = f(x(t)) ; x(0) = \bar{x}$$

$\therefore$ ~~$x(t) = \bar{x}$~~ is a solution
for $t \geq 0$

We can linearize the dynamics
around equilibria