To specify the rates of reactions in a nonisothermal reactor, we require a model to determine the temperature of the reactor, i.e. for the reaction

$$A + B \xrightarrow{k_1} C$$

$$r = k_1(T) c_A c_B - k_{-1}(T) c_C$$

The temperature is determined by the energy balance for the reactor.

We derive the energy balance by considering an arbitrary reactor volume element, shown in Figure 6.1.
The statement of conservation of energy for this system takes the form,

\[
\left\{ \text{rate of energy accumulation} \right\} = \left\{ \text{rate of energy entering system by inflow} \right\} - \left\{ \text{rate of energy leaving system by outflow} \right\} + \left\{ \text{rate of heat added to system} \right\} + \left\{ \text{rate of work done on system} \right\}
\]  

\(1\)
In terms of the defined variables, we can write Equation 6.1 as,

$$\frac{dE}{dt} = m_0 \dot{E}_0 - m_1 \dot{E}_1 + \dot{Q} + \dot{W}$$  \hspace{1cm} (6.2)$$

in which the hat indicates an energy per unit mass.

It is convenient to split the work term into three parts: $\dot{W}_f$, the work done by the flow streams while moving material into and out of the reactor, $\dot{W}_s$, the shaft work being done by stirrers, compressors, etc., and $\dot{W}_b$, the work done when moving the system boundary.

$$\dot{W} = \dot{W}_f + \dot{W}_s + \dot{W}_b$$  \hspace{1cm} (6.3)$$
Work done by the flow streams

\[ \dot{W}_f = v_0 A_0 P_0 - v_1 A_1 P_1 = Q_0 P_0 - Q_1 P_1 \]

Figure 6.2: Flow streams entering and leaving the volume element.

We also can express the volumetric flowrate as a mass flowrate divided by the density, \( Q = m/\rho \)

\[ \dot{W}_f = m_0 \frac{P_0}{\rho_0} - m_1 \frac{P_1}{\rho_1} \]

The overall rate of work can then be expressed as

\[ \dot{W} = \dot{W}_f + \dot{W}_s + \dot{W}_b = m_0 \frac{P_0}{\rho_0} - m_1 \frac{P_1}{\rho_1} + \dot{W}_s + \dot{W}_b \quad (6.4) \]
Energy Terms

The total energy may be regarded as composed of many forms. Obvious contributions to the total energy arise from the internal, kinetic and potential energies.\(^1\)

\[ \hat{E} = \hat{U} + \hat{K} + \hat{\Phi} + \cdots \]

For our purposes in this chapter, we consider only these forms of energy. Recalling the definition of enthalpy, \(H = U + PV\), or expressed on a per-unit mass basis, \(\hat{H} = \hat{U} + P/\rho\), allows us to rewrite Equation 6.2 as

\[
\frac{d}{dt} (U + K + \Phi) = m_0 \left( \hat{H} + \hat{K} + \hat{\Phi} \right)_0 - m_1 \left( \hat{H} + \hat{K} + \hat{\Phi} \right)_1 + \dot{Q} + \dot{W}_s + \dot{W}_b \tag{6.5}
\]

\(^1\)In some cases one might need to consider also electrical and magnetic energies. For example, we might consider the motion of charged ionic species between the plates in a battery cell.
Since the batch reactor has no flow streams Equation 6.5 reduces to

$$
\frac{d}{dt}(U + K + \Phi) = \dot{Q} + \dot{W}_s + \dot{W}_b
$$

(6.6)

In chemical reactors, we normally assume the internal energy is the dominant contribution and neglect the kinetic and potential energies. Normally we neglect the work done by the stirrer, unless the mixture is highly viscous and the stirring operation draws significant power [3]. Neglecting kinetic and potential energies and shaft work yields

$$
\frac{dU}{dt} + P \frac{dV_R}{dt} = \dot{Q}
$$

(6.7)

in which \(\dot{W}_b = -PdV_R/dt\).
Batch reactor in terms of enthalpy

It is convenient to use enthalpy rather than internal energy in the subsequent development. Taking the differential of the definition of enthalpy gives for $V = V_R$

$$dH = dU + PdV_R + V_RdP$$

Forming the time derivatives and substitution into Equation 6.7 gives

$$\frac{dH}{dt} - V_R \frac{dP}{dt} = \dot{Q} \quad (6.8)$$
Effect of changing $T$, $P$, $n_j$

For single-phase systems, we consider the enthalpy as a function of temperature, pressure and number of moles, and express its differential as

$$dH = \left( \frac{\partial H}{\partial T} \right)_{P,n_j} dT + \left( \frac{\partial H}{\partial P} \right)_{T,n_j} dP + \sum_j \left( \frac{\partial H}{\partial n_j} \right)_{T,P,n_k} dn_j$$  \hspace{1cm} (6.9)

The first partial derivative is the definition of the heat capacity, $C_P$.

$$C_P = V_R \rho \hat{C}_P$$

The second partial derivative can be expressed as

$$\left( \frac{\partial H}{\partial P} \right)_{T,n_j} = V - T \left( \frac{\partial V}{\partial T} \right)_{P,n_j} = V(1 - \alpha T)$$

in which $\alpha = (1/V)(\partial V/\partial T)_{P,n_j}$ is the coefficient of expansion of the mixture. The final partial derivatives are the partial molar enthalpies, $\bar{H}_j$

$$\left( \frac{\partial H}{\partial n_j} \right)_{T,P,n_k} = \bar{H}_j$$

so Equation 6.9 can be written compactly as

$$dH = V_R \rho \hat{C}_P dT + (1 - \alpha T)V_R dP + \sum_j \bar{H}_j dn_j$$  \hspace{1cm} (6.10)
Meanwhile, back in the batch reactor

Forming the time derivatives from this expression and substituting into Equation 6.8 gives

\[
V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha T V_R \frac{dP}{dt} + \sum_j \bar{H}_j \frac{dn_j}{dt} = \dot{Q}
\]  

(6.11)

We note that the material balance for the batch reactor is

\[
\frac{dn_j}{dt} = R_j V_R = \sum_{i=1}^{n_r} \nu_{ij} r_i V_R, \quad j = 1, \ldots, n_s
\]  

(6.12)

which upon substitution into Equation 6.11 yields

\[
V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha T V_R \frac{dP}{dt} = -\sum_i \Delta H_{Ri} r_i V_R + \dot{Q}
\]  

(6.13)

in which \(\Delta H_{Ri}\) is the heat of reaction

\[
\Delta H_{Ri} = \sum_j \nu_{ij} \bar{H}_j
\]  

(6.14)
We now consider several special cases. If the reactor operates at constant pressure \((dP/dt = 0)\) or the fluid is incompressible \((\alpha = 0)\), then Equation 6.13 reduces to

Incompressible-fluid or constant-pressure reactor.

\[ V_R \rho \hat{C}_P \frac{dT}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \dot{Q} \]  

(6.15)
A plethora of special cases – constant volume

Change from $T, P, n_j$ to $T, V, n_j$ by considering $P$ to be a function of $T, V (V = V_R), n_j$

$$dP = \left( \frac{\partial P}{\partial T} \right)_{V, n_j} dT + \left( \frac{\partial P}{\partial V} \right)_{T, n_j} dV + \sum_j \left( \frac{\partial P}{\partial n_j} \right)_{T, V, n_k} dn_j$$

For reactor operation at constant volume, $dV = 0$, and forming time derivatives and substituting into Equation 6.11 gives

$$\left[ V_R \rho \hat{C}_P - \alpha TV_R \left( \frac{\partial P}{\partial T} \right)_{V, n_j} \right] \frac{dT}{dt} + \sum_j \left[ H_j - \alpha TV_R \left( \frac{\partial P}{\partial n_j} \right)_{T, V, n_k} \right] \frac{dn_j}{dt} = \dot{Q}$$

We note that the first term in brackets is $C_V = V_R \rho \hat{C}_V$ (see Exercise 6.23)

$$V_R \rho \hat{C}_V = V_R \rho \hat{C}_P - \alpha TV_R \left( \frac{\partial P}{\partial T} \right)_{V, n_j}$$

The pressure derivative with respect to the moles can be shown to be (see Exercise 6.23)

$$\left( \frac{\partial P}{\partial n_j} \right)_{T, V, n_k \neq j} = \frac{V_j}{V \kappa_T}$$

in which $\kappa_T = -(1/V)(\partial V/\partial P)_{T, n_j}$ is the isothermal compressibility of the mixture, and $V_j$ is the partial molar volume.
Substitution of these two thermodynamic relations and the material balance yields the energy balance for the constant-volume batch reactor

\[
V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i \left( \Delta H_{Ri} - \frac{\alpha}{\kappa_T} T \Delta V_{Ri} \right) r_i V_R + \dot{Q}
\]  

(6.16)

If we consider an ideal gas, it is straightforward to calculate \( \alpha T = 1 \), \( \kappa_T P = 1 \), and \( \Delta V_{Ri} = \bar{\nu}_i (RT / P) \), where \( \bar{\nu}_i = \sum_j \nu_{ij} \). Substitution into the constant-volume energy balance gives

\[
V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i \left( \Delta H_{Ri} - RT \bar{\nu}_i \right) r_i V_R + \dot{Q}
\]  

(6.17)

where \( \bar{\nu}_i = \sum_j \nu_{ij} \).
Example 6.1: Constant-pressure versus constant-volume reactors

Consider the following two well-mixed, adiabatic, gas-phase batch reactors for the elementary and irreversible decomposition of A to B,

\[ \text{A} \xrightarrow{k} \text{2B} \]

**Reactor 1:** The reactor volume is held constant (reactor pressure therefore changes).

**Reactor 2:** The reactor pressure is held constant (reactor volume therefore changes).

Both reactors are charged with pure A at 1.0 atm and \( k \) has the usual Arrhenius activation energy dependence on temperature,

\[ k(T) = k_0 \exp(-E/T) \]

The heat of reaction, \( \Delta H_R \), and heat capacity of the mixture, \( \hat{C}_P \), may be assumed constant over the composition and temperature range expected.

Write the material and energy balances for these two reactors. Which reactor converts the reactant more quickly?
The material balance is

\[
\frac{d(c_A V_R)}{dt} = R_A V_R
\]

Substituting in the reaction-rate expression, \( r = k(T)c_A \), and using the number of moles of A, \( n_A = c_A V_R \) yields

\[
\frac{dn_A}{dt} = -k(T)n_A \quad (6.20)
\]

Notice the temperature dependence of \( k(T) \) prevents us from solving this differential equation immediately.

We must solve it simultaneously with the energy balance, which provides the information for how the temperature changes.
The energy balances for the two reactors are not the same. We consider first the constant-volume reactor. For the $A \rightarrow 2B$ stoichiometry, we substitute the rate expression and $\bar{\nu} = 1$ into Equation 6.17 to obtain

$$C_V \frac{dT}{dt} = - (\Delta H_R - RT) k n_A$$

in which $C_V = V_R \rho \hat{C}_V$ is the total constant-volume heat capacity.
The energy balance for the constant-pressure case follows from Equation 6.15

\[ C_P \frac{dT}{dt} = -\Delta H_R k n_A \]

in which \( C_P = V_R \rho \hat{C}_P \) is the total constant-pressure heat capacity. For an ideal gas, we know from thermodynamics that the two total heat capacities are simply related,

\[ C_V = C_P - nR \] (6.21)
Express all moles in terms of \( n_A \)

Comparing the production rates of A and B produces

\[
2n_A + n_B = 2n_{A_0} + n_{B_0}
\]

Because there is no B in the reactor initially, subtracting \( n_A \) from both sides yields for the total number of moles

\[
n = n_A + n_B = 2n_{A_0} - n_A
\]
The two energy balances

Substitution of the above and Equation 6.21 into the constant-volume case yields

$$\frac{dT}{dt} = -\frac{(\Delta H_R - RT) kn_A}{C_P - (2n_{A0} - n_A)R} \quad \text{constant volume} \quad (6.22)$$

and the temperature differential equation for the constant-pressure case is

$$\frac{dT}{dt} = -\frac{\Delta H_R kn_A}{C_P} \quad \text{constant pressure} \quad (6.23)$$
So who’s faster?

We see by comparing Equations 6.22 and 6.23 that the numerator in the constant-volume case is larger because $\Delta H_R$ is negative and the positive $RT$ is subtracted. We also see the denominator is smaller because $C_P$ is positive and the positive $nR$ is subtracted. Therefore the time derivative of the temperature is larger for the constant-volume case. The reaction proceeds more quickly in the constant-volume case. The constant-pressure reactor is expending work to increase the reactor size, and this work results in a lower temperature and slower reaction rate compared to the constant-volume case.
Example 6.2: Liquid-phase batch reactor

The exothermic elementary liquid-phase reaction

\[ A + B \xrightarrow{k} C, \quad r = k c_A c_B \]

is carried out in a batch reactor with a cooling coil to keep the reactor isothermal at 27°C. The reactor is initially charged with equal concentrations of A and B and no C, \( c_{A0} = c_{B0} = 2.0 \text{ mol/L} \), \( c_{C0} = 0 \).

1. How long does it take to reach 95% conversion?
2. What is the total amount of heat (kcal) that must be removed by the cooling coil when this conversion is reached?
3. What is the maximum rate at which heat must be removed by the cooling coil (kcal/min) and at what time does this maximum occur?
4. What is the adiabatic temperature rise for this reactor and what is its significance?

Additional data:

- Rate constant, \( k = 0.01725 \text{ L/mol·min} \), at 27°C
- Heat of reaction, \( \Delta H_R = -10 \text{ kcal/mol A} \), at 27°C
- Partial molar heat capacities, \( \overline{C}_P_A = \overline{C}_P_B = 20 \text{ cal/mol·K} \), \( \overline{C}_P_C = 40 \text{ cal/mol K} \)
- Reactor volume, \( V_R = 1200 \text{ L} \)
Assuming constant density, the material balance for component A is

\[ \frac{dc_A}{dt} = -k c_A c_B \]

The stoichiometry of the reaction, and the material balance for B gives

\[ c_A - c_B = c_{A0} - c_{B0} = 0 \]

or \( c_A = c_B \). Substitution into the material balance for species A gives

\[ \frac{dc_A}{dt} = -k c_A^2 \]

Separation of variables and integration gives

\[ t = \frac{1}{k} \left[ \frac{1}{c_A} - \frac{1}{c_{A0}} \right] \]

Substituting \( c_A = 0.05c_{A0} \) and the values for \( k \) and \( c_{A0} \) gives

\[ t = 551 \text{ min} \]
We assume the incompressible-fluid energy balance is accurate for this liquid-phase reactor. If the heat removal is manipulated to maintain constant reactor temperature, the time derivative in Equation 6.15 vanishes leaving

\[ \dot{Q} = \Delta H_R r V_R \]  

Substituting \( dc_A/dt = -r \) and multiplying through by \( dt \) gives

\[ dQ = -\Delta H_R V_R dc_A \]

Integrating both sides gives

\[ Q = -\Delta H_R V_R (c_A - c_{A0}) = -2.3 \times 10^4 \text{ kcal} \]

Substituting \( r = k c_A^2 \) into Equation 6.24 yields

\[ \dot{Q} = \Delta H_R k c_A^2 V_R \]

The right-hand side is a maximum in absolute value (note it is a negative quantity) when \( c_A \) is a maximum, which occurs for \( c_A = c_{A0} \), giving

\[ \dot{Q}_{\text{max}} = \Delta H_R k c_{A0}^2 V_R = -828 \text{ kcal/min} \]
The adiabatic temperature rise is calculated from the energy balance without the heat-transfer term

\[ V_R \rho \hat{C}_P \frac{dT}{dt} = -\Delta H_R r V_R \]

Substituting the material balance \( dn_A/dt = -rV_R \) gives

\[ V_R \rho \hat{C}_P dT = \Delta H_R dn_A \quad (6.25) \]

Because we are given the partial molar heat capacities

\[ \overline{C}_{Pj} = \left( \frac{\partial \overline{H}_j}{\partial T} \right)_{P,n_k} \]

it is convenient to evaluate the total heat capacity as

\[ V_R \rho \hat{C}_P = \sum_{j=1}^{n_s} \overline{C}_{Pj} n_j \]

For a batch reactor, the number of moles can be related to the reaction extent by \( n_j = n_{j0} + \nu_j \varepsilon \), so we can express the right-hand side of the previous equation as

\[ \sum_{j=1}^{n_s} \overline{C}_{Pj} n_j = \sum_j \overline{C}_{Pj} n_{j0} + \varepsilon \Delta C_P \]
in which $\Delta C_P = \sum_j \nu_j \bar{C}_{Pj}$. If we assume the partial molar heat capacities are independent of temperature and composition we have $\Delta C_P = 0$ and

$$V_R \rho \hat{C}_P = \sum_{j=1}^{n_s} \bar{C}_{Pj} n_{j0}$$

Integrating Equation 6.25 with constant heat capacity gives

$$\Delta T = \frac{\Delta H_R}{\sum_j \bar{C}_{Pj} n_{j0}} \Delta n_A$$

The maximum temperature rise corresponds to complete conversion of the reactants and can be computed from the given data

$$\Delta T_{\text{max}} = \frac{-10 \times 10^3 \text{ cal/mol}}{2(2 \text{ mol/L})(20 \text{ cal/mol K})(0 - 2 \text{ mol/L})}$$

$$\Delta T_{\text{max}} = 250 \text{ K}$$

The adiabatic temperature rise indicates the potential danger of a coolant system failure. In this case the reactants contain enough internal energy to raise the reactor temperature by 250 K.
\[
\frac{d}{dt} (U + K + \Phi) = m_0 \left( \hat{H} + \hat{K} + \hat{\Phi} \right)_0 - m_1 \left( \hat{H} + \hat{K} + \hat{\Phi} \right)_1 + Q + \dot{W}_s + \dot{W}_b
\]

We assume that the internal energy is the dominant contribution to the total energy and take the entire reactor contents as the volume element. We denote the feed stream with flowrate \( Q_f \), density \( \rho_f \), enthalpy \( \hat{H}_f \), and component \( j \) concentration \( c_{jf} \). The outflow stream is flowing out of a well-mixed reactor and its intensive properties are therefore assumed the same as the reactor contents. Its flowrate is denoted \( Q \). Writing Equation 6.5 for this reactor gives,

\[
\frac{dU}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} + \dot{W}_s + \dot{W}_b \tag{6.26}
\]
If we neglect the shaft work

\[
\frac{dU}{dt} + P \frac{dV_R}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q}
\]

or if we use the enthalpy rather than internal energy \((H = U + PV)\)

\[
\frac{dH}{dt} - V_R \frac{dP}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q}
\]

(6.28)
The Enthalpy change

For a single-phase system we consider the change in $H$ due to changes in $T, P, n_j$

$$dH = V_R \rho \hat{C}_P dT + (1 - \alpha T)V_R dP + \sum_j \overline{H}_j dn_j$$

Substitution into Equation 6.28 gives

$$V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha TV_R \frac{dP}{dt} + \sum_j \overline{H}_j \frac{dn_j}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} \tag{6.29}$$

The material balance for the CSTR is

$$\frac{dn_j}{dt} = Q_f c_{jf} - Q c_j + \sum_i \nu_{ij} r_i V_R \tag{6.30}$$

Substitution into Equation 6.29 and rearrangement yields

$$V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha TV_R \frac{dP}{dt} = -\sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q} \tag{6.31}$$
Again, a variety of important special cases may be considered. These are listed in Table 6.8 in the chapter summary. A common case is the liquid-phase reaction, which usually is well approximated by the incompressible-fluid equation,

\[
V_R \rho \hat{C}_P \frac{dT}{dt} = -\sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q}
\]  

(6.32)

In the next section we consider further simplifying assumptions that require less thermodynamic data and yield useful approximations.
Neglect kinetic and potential energies

\[ \frac{dU}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} + \dot{W}_s + \dot{W}_b \]

Neglect shaft work

\[ \frac{dU}{dt} + P \frac{dV_R}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} \]
\[ \frac{dH}{dt} - V_R \frac{dP}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} \]

Single phase

\[ V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha TV_R \frac{dP}{dt} + \sum_j \overline{H}_j \frac{dn_j}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} \]
\[ V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha TV_R \frac{dP}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q} \]

a. Incompressible-fluid or constant-pressure reactor

\[ V_R \rho \hat{C}_P \frac{dT}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q} \]
b. Constant-volume reactor

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i (\Delta H_{Ri} - \frac{\alpha}{\kappa T} T \Delta V_{Ri}) r_i V_R + \sum_j c_{jf} Q_f (\bar{H}_{jf} - \bar{H}_j) + \frac{\alpha}{\kappa T} T \sum_j V_j (c_{jf} Q_f - c_j Q) + \dot{Q} \]

b.1 Constant-volume reactor, ideal gas

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i (\Delta H_{Ri} - RT \bar{\nu}_i) r_i V_R + \sum_j c_{jf} Q_f (\bar{H}_{jf} - \bar{H}_j) + RT \sum_j (c_{jf} Q_f - c_j Q) + \dot{Q} \]

c. Steady state, constant \( \hat{C}_P, P = P_f \)

\[ - \sum_i \Delta H_{Ri} r_i V_R + Q_f \rho_f \hat{C}_P (T_f - T) + \dot{Q} = 0 \]

**Table:** Energy balances for the CSTR.
If the CSTR is at steady state, the time derivatives in Equations 6.30 and 6.31 can be set to zero yielding,

\[ Q_f c_{jf} - Q c_j + \sum_i \nu_{ij} r_i V_R = 0 \]  \hfill (6.33)

\[- \sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q} = 0 \]  \hfill (6.34)

Equations 6.33 and 6.34 provide \(n_s + 1\) algebraic equations that can be solved simultaneously to obtain the steady-state concentrations and temperature in the CSTR. Note that the heats of reaction \(\Delta H_{Ri}\) are evaluated at the reactor temperature and composition.
If the heat capacity of the liquid phase does not change significantly with composition or temperature, possibly because of the presence of a large excess of a nonreacting solvent, and we neglect the pressure effect on enthalpy, which is normally small for a liquid, we obtain

\[
\bar{H}_{jf} - \bar{H}_j = \bar{C}_{Pj} (T_f - T)
\]

Substitution into Equation 6.34 gives

\[
- \sum_i r_i \Delta H_{Ri} V_R + Q_f \rho_f \hat{C}_P (T_f - T) + \dot{Q} = 0
\]  

(6.35)
Example 6.3: Temperature control in a CSTR

An aqueous solution of species A undergoes the following elementary reaction in a 2000 L CSTR

\[
A \xrightarrow{k_1} \overset{k}{\underset{k_{-1}}{\rightleftharpoons}} R \quad \Delta H_R = -18 \text{ kcal/mol}
\]

The feed concentration, $C_{Af}$, is 4 mol/L and feed flowrate, $Q_f$, is 250 L/min.
The reaction-rate constants have been determined experimentally

\[ k_1 = 3 \times 10^7 e^{-5838/T} \text{ min}^{-1} \]
\[ K_1 = 1.9 \times 10^{-11} e^{9059/T} \]

1. At what temperature must the reactor be operated to achieve 80% conversion?

2. What are the heat duties of the two heat exchangers if the feed enters at 25°C and the product is to be withdrawn at this temperature? The heat capacity of feed and product streams can be approximated by the heat capacity of water, \( \hat{C}_P = 1 \text{ cal/g K} \).
The steady-state material balances for components A and R in a constant-density CSTR are

\[
Q(c_{Af} - c_A) - rV_R = 0 \\
Q(c_{Rf} - c_R) + rV_R = 0
\]

Adding these equations and noting \(c_{Rf} = 0\) gives

\[c_R = c_{Af} - c_A\]

Substituting this result into the rate expression gives

\[r = k_1(c_A - \frac{1}{K_1}(c_{Af} - c_A))\]

Substitution into the material balance for A gives

\[Q(c_{Af} - c_A) - k_1(c_A - \frac{1}{K_1}(c_{Af} - c_A))V_R = 0\] \hspace{1cm} (6.36)

If we set \(c_A = 0.2c_{Af}\) to achieve 80% conversion, we have one equation and one unknown, \(T\), because \(k_1\) and \(K_1\) are given functions of temperature. Solving this equation numerically gives

\[T = 334 \text{ K}\]
Because the reaction is reversible, we do not know if 80% conversion is achievable for any temperature when we attempt to solve Equation 6.36. It may be valuable to first make a plot of the conversion as a function of reactor temperature. If we solve Equation 6.36 for $c_A$, we have

$$c_A = \frac{Q/V_R + k_1/K_1}{Q/V_R + k_1(1 + 1/K_1)} c_{Af}$$

or for $x_A = 1 - c_A/c_{Af}$

$$x_A = \frac{k_1}{Q/V_R + k_1(1 + 1/K_1)} = \frac{k_1 \tau}{1 + k_1 \tau(1 + 1/K_1)}$$
We see that the conversion 80% is just reachable at 334 K, and that for any conversion lower than this value, there are two solutions.
A simple calculation for the heat-removal rate required to bring the reactor outflow stream from 334 K to 298 K gives

\[
\dot{Q}_2 = Q_f \rho \hat{C}_P \Delta T \\
= (250 \text{ L/min})(1000 \text{ g/L})(1 \text{ cal/g K})(298 - 334 \text{ K}) \\
= -9 \times 10^3 \text{ kcal/min}
\]

Applying Equation 6.35 to this reactor gives

\[
\dot{Q}_1 = k_1 \left( c_A - \frac{1}{K_1} (c_{Af} - c_A) \right) \Delta H_R V_R - Q_f \rho \hat{C}_P (T_f - T) \\
= -5.4 \times 10^3 \text{ kcal/min}
\]
Steady-State Multiplicity

- The coupling of the material and energy balances for the CSTR can give rise to some surprisingly complex and interesting behavior.
- Even the steady-state solution of the material and energy balances holds some surprises.
- In this section we explore the fact that the steady state of the CSTR is not necessarily unique.
- As many as three steady-state solutions to the material and energy balances may exist for even the simplest kinetic mechanisms.
- This phenomenon is known as steady-state multiplicity.
We introduce this topic with a simple example [5]. Consider an adiabatic, constant-volume CSTR with the following elementary reaction taking place in the liquid phase

\[ A \overset{k}{\longrightarrow} B \]

We wish to compute the steady-state reactor conversion and temperature. The data and parameters are listed in Table 6.1.
<table>
<thead>
<tr>
<th>Parameter</th>
<th>Value</th>
<th>Units</th>
</tr>
</thead>
<tbody>
<tr>
<td>$T_f$</td>
<td>298</td>
<td>K</td>
</tr>
<tr>
<td>$T_m$</td>
<td>298</td>
<td>K</td>
</tr>
<tr>
<td>$\hat{C}_P$</td>
<td>4.0</td>
<td>kJ/kg K</td>
</tr>
<tr>
<td>$c_{Af}$</td>
<td>2.0</td>
<td>kmol/m$^3$</td>
</tr>
<tr>
<td>$k_m$</td>
<td>0.001</td>
<td>min$^{-1}$</td>
</tr>
<tr>
<td>$E$</td>
<td>$8.0 \times 10^3$</td>
<td>K</td>
</tr>
<tr>
<td>$\rho$</td>
<td>$10^3$</td>
<td>kg/m$^3$</td>
</tr>
<tr>
<td>$\Delta H_R$</td>
<td>$-3.0 \times 10^5$</td>
<td>kJ/kmol</td>
</tr>
<tr>
<td>$U^o$</td>
<td>0</td>
<td></td>
</tr>
</tbody>
</table>

**Table 6.1:** Parameter values for multiple steady states.
The material balance for component A is

\[
\frac{d(c_A V_R)}{dt} = Q_f c_{Af} - Q c_A + R_A V_R
\]

The production rate is given by

\[R_A = -k(T)c_A\]

For the steady-state reactor with constant-density, liquid-phase streams, the material balance simplifies to

\[0 = c_{Af} - (1 + k\tau)c_A\] (6.37)
Equation 6.37 is one nonlinear algebraic equation in two unknowns: $c_A$ and $T$. The temperature appears in the rate-constant function,

$$k(T) = k_m e^{-E(1/T - 1/T_m)}$$

Now we write the energy balance. We assume the heat capacity of the mixture is constant and independent of composition and temperature.
Energy balance

b. Constant-volume reactor

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i (\Delta H_{Ri} - \frac{\alpha}{\kappa T} T \Delta V_{Ri}) r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \frac{\alpha}{\kappa T} T \sum_j V_j (c_{jf} Q_f - c_j Q) + \dot{Q} \]

b.1 Constant-volume reactor, ideal gas

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i (\Delta H_{Ri} - RT \bar{v}_i) r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + RT \sum_j (c_{jf} Q_f - c_j Q) + \dot{Q} \]

c. Steady state, constant \( \hat{C}_P, P = P_f \)

\[- \sum_i \Delta H_{Ri} r_i V_R + Q_f \rho_f \hat{C}_P (T_f - T) + \dot{Q} = 0 \]

Table: Energy balances for the CSTR.
Energy balance

We assume the heat capacity of the mixture is constant and independent of composition and temperature. With these assumptions, the steady-state energy balance reduces to

\[ 0 = -kc_A \Delta H R V_R + Q_f \rho_f \hat{C}_P (T_f - T) + U^o A (T_a - T) \]

Dividing through by \( V_R \) and noting \( U^o = 0 \) for the adiabatic reactor gives

\[ 0 = -kc_A \Delta H R + \frac{C_{Ps}}{\tau} (T_f - T) \]

(6.38)

in which \( C_{Ps} = \rho_f \hat{C}_P \), a heat capacity per volume.
Two algebraic equations, two unknowns

\[ 0 = c_{Af} - (1 + k(T)\tau)c_A \]

\[ 0 = -k(T)c_A\Delta H_R + \frac{C_{Ps}}{\tau}(T_f - T) \]

The solution of these two equations for \( c_A \) and \( T \) provide the steady-state CSTR solution. The parameters appearing in the problem are: \( c_{Af}, T_f, \tau, C_{Ps}, k_m, T_m, E, \Delta H_R \). We wish to study this solution as a function of one of these parameters, \( \tau \), the reactor residence time.
Let's compute

- Set $\Delta H_R = 0$ to model the isothermal case. Find $c_A(\tau)$ and $T(\tau)$ as you vary $\tau$, $0 \leq \tau \leq 1000$ min.
- We will need the following trick later. Switch the roles of $c_A$ and $\tau$ and find $\tau(c_A)$ and $T(c_A)$ versus $c_A$ for $0 \leq c_A \leq c_{Af}$.
- It doesn't matter which is the parameter and which is the unknown, you can still plot $c_A(\tau)$ and $T(\tau)$. Check that both approaches give the same plot.
- For the isothermal reactor, we already have shown that

$$c_A = \frac{c_{Af}}{1 + k\tau}, \quad x = \frac{k\tau}{1 + k\tau}$$
Exothermic (and endothermic) cases

- Set $\Delta H_R = -3 \times 10^5$ kJ/kmol and solve again. What has happened?
- Fill in a few more $\Delta H_R$ values and compare to Figures 6.5–6.6.
- Set $\Delta H_R = +5 \times 10^4$ kJ/kmol and try an endothermic case.
Figure 6.5: Steady-state conversion versus residence time for different values of the heat of reaction ($\Delta H_R \times 10^{-4}$ kJ/kmol).
Figure 6.6: Steady-state temperature versus residence time for different values of the heat of reaction ($\Delta H_R \times 10^{-4}$ kJ/kmol).
Note that if the heat of reaction is more exothermic than $-10 \text{ kJ/kmol}$, there is a range of residence times in which there is not one but several steady-state solutions, three solutions in this case.

The reactor is said to exhibit **steady-state multiplicity** for these values of residence time.

The points at which the steady-state curves turn are known as **ignition** and **extinction** points.
Figure 6.7: Steady-state conversion versus residence time for $\Delta H_R = -3 \times 10^5$ kJ/kmol; ignition and extinction points.
Figure 6.8: Steady-state temperature versus residence time for $\Delta H_R = -3 \times 10^5$ kJ/kmol; ignition and extinction points.
Consider a small value of residence time, 10 min, at low conversion of A and low temperature. If the feed flowrate were decreased slightly ($\tau$ increased), there would be a small upset and the reactor would increase in conversion and temperature as it approached the new steady state at the new residence time.

Consider the situation at the ignition point, however, $\tau = 30.9$ min at $x = 0.09$ and $T = 311$ K. If there is a small decrease in feed flowrate there is no steady state near the temperature and concentration of the reactor. A large release of heat occurs and the reactor ignites and moves to the steady state near $x = 1$ and $T = 448$ K.

A reactor operating near the extinction point can exhibit the opposite phenomenon. A small increase in feed flowrate causes the residence time to decrease enough so that no steady-state solution exists near the current temperature and concentration. A rapid drop in temperature and increase in concentration of A occurs as the reactor approaches the new steady state.
We next discuss why some steady states are stable and others are unstable. This discussion comes in two parts. First we present a plausibility argument and develop some physical intuition by constructing and examining van Heerden diagrams [6]. The text also presents a rigorous mathematical argument, which has wide applicability in analyzing the stability of any system described by differential equations.

**Dynamic Model**

We examine the stability numerically by solving the dynamic model.

\[
\frac{dc_A}{dt} = \frac{c_A f - c_A}{\tau} - k c_A
\]

\[
\frac{dT}{dt} = \frac{U^o A}{V_R C_p s} (T_a - T) + \frac{T_f - T}{\tau} - \frac{\Delta H_R}{C_p s} k c_A
\]
Figure 6.9: Steady-state temperature versus residence time for $\Delta H_R = -3 \times 10^5$ kJ/kmol.
Heat generate and heat removal

If we substitute the the mass balance for $c_A$ into the energy balance, we obtain one equation for one unknown, $T$,

$$0 = -\frac{k}{1 + k\tau}cAf\Delta H_R + \frac{C_{Ps}}{\tau}(T_f - T)$$

We call the first term the heat-generation rate, $\dot{Q}_g$. We call the second term the heat-removal rate, $\dot{Q}_r$,

$$\dot{Q}_g = -\frac{k(T)}{1 + k(T)\tau}cAf\Delta H_R, \quad \dot{Q}_r = \frac{C_{Ps}}{\tau}(T - T_f)$$

in which we emphasize the temperature dependence of the rate constant,

$$k(T) = k_m e^{-E(1/T - 1/T_m)}$$
Obviously we have a steady-state solution when these two quantities are equal. Consider plotting these two functions as $T$ varies.
The heat-removal rate is simply a straight line with slope $C_{Ps}/\tau$.
The heat-generation rate is a nonlinear function that is asymptotically constant at low temperatures ($k(T)$ much less than one) and high temperatures ($k(T)$ much greater than one).
These two functions are plotted next for $\tau = 1.79$ min.
Van Heerden diagram

Figure 6.10: Rates of heat generation and removal for $\tau = 1.79$ min.
Changing the residence time

Notice the two intersections of the heat-generation and heat-removal functions corresponding to steady states A and E. If we decrease the residence time slightly, the slope of the heat-removal line increases and the intersection corresponding to point A shifts slightly. Because the two curves are just tangent at point E, however, the solution at point E disappears, another indicator that point E is an extinction point.
The other view of changing the residence time.
If we were to increase the reactor temperature slightly, we would be to the right of point A in Figure 6.10. To the right of A we notice that the heat-removal rate is larger than the heat-generation rate. That causes the reactor to cool, which moves the temperature back to the left. In other words, the system responds by resisting our applied perturbation. Similarly, consider a decrease to the reactor temperature. To the left of point A, the heat-generation rate is larger than the heat-removal rate causing the reactor to heat up and move back to the right. Point A is a stable steady state because small perturbations are rejected by the system.
Consider next the points on the middle branch. Figure 6.11 displays the heat-generation and heat-removal rates for points B, D and F, $\tau = 15$ min.
Figure 6.11: Rates of heat generation and removal for $\tau = 15$ min.
Consider next the points on the middle branch. Figure 6.11 displays the heat-generation and heat-removal rates for points B, D and F, $\tau = 15$ min.

Point B on the lower branch is stable as was point A. Point F on the upper branch also is stable because the slope of the heat-generation rate is smaller than the heat-removal rate at point F.

At point D, however, the slope of the heat-generation rate is larger than the heat-removal rate.

For point D, increasing temperature causes heat generation to be larger than heat removal, and decreasing temperature causes heat generation to be smaller than heat removal. Both of these perturbations are amplified by the system at point D, and this solution is unstable.

All points on the middle branch are similar to point D.
Large residence time

\[ \tau (\text{min}) \]

\[ T (\text{K}) \]

Points labeled A, B, D, E, F, G on the graph.
Next observe the heat-generation and heat-removal rates for $\tau = 30.9$ min.
Notice that point G on the upper branch is stable and point C, the ignition point, is similar to extinction point E, perturbations in one direction are rejected, but in the other direction they are amplified.
I will not cover this section in lecture. Please read the text.
A simple mechanical analogy

You may find it helpful to draw an analogy between the chemical reactor with multiple steady states and simple mechanical systems that exhibit the same behavior. Consider a marble on a track in a gravitational field as depicted in Figure 6.15.
A simple mechanical analogy

Figure 6.15: Marble on a track in a gravitational field; point A is the unique, stable steady state.
Based on our physical experience with such systems we conclude immediately that the system has a single steady state, position A, which is asymptotically stable. If we expressed Newton’s laws of motion for this system, and linearized the model at point A, we would expect to see eigenvalues with negative real part and nonzero imaginary part because the system exhibits a decaying oscillation back to the steady-state position after a perturbation. The oscillation decays because of the friction between the marble and the track.
Multiple steady states

Now consider the track depicted in Figure 6.16.
A simple mechanical analogy

Figure: Marble on a track in a gravitational field; point A is the unique, stable steady state.

Figure 6.16: Marble on a track with three steady states; points A and C are stable, and point B is unstable.
Here we have three steady states, the three positions where the tangent curve to the track has zero slope. This situation is analogous to the chemical reactor with multiple steady states. The steady states A and C are obviously stable and B is unstable. Perturbations from point B to the right are attracted to steady-state C and perturbations to the left are attracted to steady-state A. The significant difference between the reactor and marble systems is that the marble decays to steady state in an oscillatory fashion, and the reactor, with its zero imaginary eigenvalues, returns to the steady state without overshoot or oscillation.
Now consider the track depicted in Figure 6.17.
A simple mechanical analogy

Figure: Marble on a track in a gravitational field; point A is the unique, stable steady state.

Figure: Marble on a track with three steady states; points A and C are stable, and point B is unstable.

Figure 6.17: Marble on a track with an ignition point (A) and a stable steady state (C).
We have flattened the track between points A and B in Figure 6.16 so there is just a single point of zero slope, now marked point A. Point A now corresponds to a reactor ignition point as shown in Figures 6.7 and 6.8. Small perturbations push the marble over to the only remaining steady state, point C, which remains stable.
The dynamic behavior of the CSTR can be more complicated than multiple steady states with ignition, extinction and hysteresis. In fact, at a given operating condition, all steady states may be unstable and the reactor may exhibit sustained oscillations or limit cycles. Consider the same simple kinetic scheme as in the previous section,

\[ A \xrightarrow{k} B \]

but with the following parameter values.
Table 6.3: Parameter values for limit cycles.

<table>
<thead>
<tr>
<th>Param.</th>
<th>Value</th>
<th>Units</th>
</tr>
</thead>
<tbody>
<tr>
<td>$T_f$</td>
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<td>K</td>
</tr>
<tr>
<td>$T_m$</td>
<td>298</td>
<td>K</td>
</tr>
<tr>
<td>$\hat{C}_P$</td>
<td>4.0</td>
<td>kJ/kg K</td>
</tr>
<tr>
<td>$c_{Af}$</td>
<td>2.0</td>
<td>kmol/m$^3$</td>
</tr>
<tr>
<td>$k_m(T_m)$</td>
<td>0.004</td>
<td>min$^{-1}$</td>
</tr>
<tr>
<td>$E$</td>
<td>$1.5 \times 10^4$</td>
<td>K</td>
</tr>
<tr>
<td>$\rho$</td>
<td>$10^3$</td>
<td>kg/m$^3$</td>
</tr>
<tr>
<td>$\Delta H_R$</td>
<td>$-2.2 \times 10^5$</td>
<td>kJ/kmol</td>
</tr>
<tr>
<td>$U^oA/V_R$</td>
<td>340</td>
<td>kJ/(m$^3$ min K)</td>
</tr>
</tbody>
</table>

Note: the last line of this table is missing in the first printing!
Notice that the activation energy in Table 6.3 is significantly larger than in Table 6.1.
Even more twisted steady-state behavior

If we compute the solutions to the steady-state mass and energy balances with these new values of parameters, we obtain the results displayed in the next Figures.
Figure 6.18: Steady-state conversion versus residence time.
If we replot these results using a log scaling to stretch out the $\times$ axis, we obtain the results in Figures 6.20 and 6.21.
Figure 6.20: Steady-state conversion versus residence time — log scale.
Notice the steady-state solution curve has become a bit deformed and the simple s-shaped multiplicities in Figures 6.7 and 6.8 have taken on a mushroom shape with the new parameters.
We have labeled points A–F on the steady-state curves. Table 6.4 summarizes the locations of these points in terms of the residence times, the steady-state conversions and temperatures, and the eigenvalue of the Jacobian with largest real part.

<table>
<thead>
<tr>
<th>Point</th>
<th>$\tau$ (min)</th>
<th>$\chi$</th>
<th>$T$ (K)</th>
<th>Re($\lambda$) (min$^{-1}$)</th>
<th>Im($\lambda$) (min$^{-1}$)</th>
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</thead>
<tbody>
<tr>
<td>A</td>
<td>11.1</td>
<td>0.125</td>
<td>305</td>
<td>0</td>
<td>0</td>
</tr>
<tr>
<td>B</td>
<td>0.008</td>
<td>0.893</td>
<td>396</td>
<td>0</td>
<td>0</td>
</tr>
<tr>
<td>C</td>
<td>19.2</td>
<td>0.970</td>
<td>339</td>
<td>$-0.218$</td>
<td>0</td>
</tr>
<tr>
<td>D</td>
<td>20.7</td>
<td>0.962</td>
<td>336</td>
<td>$-0.373$</td>
<td>0</td>
</tr>
<tr>
<td>E</td>
<td>29.3</td>
<td>0.905</td>
<td>327</td>
<td>0</td>
<td>0.159</td>
</tr>
<tr>
<td>F</td>
<td>71.2</td>
<td>0.519</td>
<td>306</td>
<td>0</td>
<td>0.0330</td>
</tr>
</tbody>
</table>

Table 6.4: Steady state and eigenvalue with largest real part at selected points in Figures 6.20 and 6.21.
Now we take a walk along the steady-state solution curve and examine the dynamic behavior of the system.
What can simulations can show us. A residence time of $\tau = 35$ min is between points E and F as shown in Table 6.4. Solving the dynamic mass and energy balances with this value of residence time produces
Figure 6.25: Conversion and temperature vs. time for $\tau = 35$ min.
We see that the solution does not approach the steady state but oscillates continuously. These oscillations are sustained; they do not damp out at large times. Notice also that the amplitude of the oscillation is large, more than 80 K in temperature and 50% in conversion.

We can obtain another nice view of this result if we plot the conversion versus the temperature rather than both of them versus time. This kind of plot is known as a phase plot or phase portrait.
Figure 6.26: Phase portrait of conversion versus temperature for feed initial condition; $\tau = 35$ min.
Time increases as we walk along the phase plot; the reactor ignites, then slowly decays, ignites again, and eventually winds onto the steady limit cycle shown in the figure. The next figure explores the effect of initial conditions.
Figure 6.27: Phase portrait of conversion versus temperature for several initial conditions; \( \tau = 35 \) min.
The trajectory starting with the feed temperature and concentration is shown again. The trajectory starting in the upper left of the figure has the feed temperature and zero concentration as its initial condition. Several other initial conditions inside the limit cycle are shown also, including starting the reactor at the unstable steady state. All of these initial conditions wind onto the same final limit cycle. We say that the limit cycle is a global attractor because all initial conditions wind onto this same solution.
If we decrease the residence time to $\tau = 30$ min, we are close to point E, where the stability of the upper steady state changes. A simulation at this residence time is shown in the next figure. Notice the amplitude of these oscillations is much smaller, and the shape is more like a pure sine wave.
Figure 6.28: Conversion and temperature vs. time for $\tau = 30$ min.
Figure 6.29: Conversion and temperature vs. time for $\tau = 72.3$ min.
As we pass point F, the steady state is again stable. A simulation near point F is shown in Figure 6.29. Notice, in contrast to point E, the amplitude of the oscillations is not small near point F.

To see how limit cycles can remain after the steady state regains its stability, consider the next figure, constructed for $\tau = 73.1$ min.
Figure 6.30: Phase portrait of conversion versus temperature at $\tau = 73.1$ min showing stable and unstable limit cycles, and a stable steady state.
The figure depicts the stable steady state, indicated by a solid circle, surrounded by an unstable limit cycle, indicated by the dashed line. The unstable limit cycle is in turn surrounded by a stable limit cycle. Note that all initial conditions outside of the stable limit cycle would converge to the stable limit cycle from the outside.

All initial conditions in the region between the unstable and stable limit cycles would converge to the stable limit cycle from the inside.

Finally, all initial conditions inside the unstable limit cycle are attracted to the stable steady state.

We have a quantitative measure of a perturbation capable of knocking the system from the steady state onto a periodic solution.
We may modify our simple mechanical system to illustrate somewhat analogous limit-cycle behavior. Consider the marble and track system.
We have three steady states; steady-state A is again stable and steady-state B is unstable. At this point we cheat thermodynamics a bit to achieve the desired behavior. Imagine the track consists of a frictionless material to the right of point B. Without friction in the vicinity of point C, the steady state is not asymptotically stable. Perturbations from point C do not return to the steady state but continually oscillate. The analogy is not perfect because a single limit cycle does not surround the unstable point C as in the chemical reactor. But the analogy may prove helpful in demystifying these kinds of reactor behaviors.

Consider why we were compelled to violate the second law to achieve sustained oscillations in the simple mechanical system but the reactor can continually oscillate without such violation.
Large topic, studied intensely by chemical engineering researchers in the 1970s–1980s.
The development of the semi-batch reactor energy balance follows directly from the CSTR energy balance derivation by setting $Q = 0$. The main results are summarized in Table 6.9 at the end of this chapter.
To derive an energy balance for the plug-flow reactor (PFR), consider the volume element

\[
\begin{align*}
Q_c^j (z + \Delta z) & = Q_c^j (z) + \dot{Q} \\
\hat{H}(z) & = \hat{H}(z + \Delta z)
\end{align*}
\]
If we write Equation 6.5 for this element and neglect kinetic and potential energies and shaft work, we obtain

$$\frac{\partial}{\partial t}(\rho \hat{U} A_c \Delta z) = m \hat{H}|_z - m \hat{H}|_{z+\Delta z} + \dot{Q}$$

in which $A_c$ is the cross-sectional area of the tube, $R$ is the tube outer radius, and $\dot{Q}$ is the heat transferred through the wall, normally expressed using an overall heat-transfer coefficient

$$\dot{Q} = U^o 2\pi R \Delta z (T_a - T)$$

Dividing by $A_c \Delta z$ and taking the limit $\Delta z \to 0$, gives

$$\frac{\partial}{\partial t}(\rho \hat{U}) = -\frac{1}{A_c} \frac{\partial}{\partial z}(Q \rho \hat{H}) + \dot{q}$$

in which $\dot{q} = (2/R)U^o(T_a - T)$ and we express the mass flowrate as $m = Q \rho$. 
Some thermodynamics occurs, and...

In unpacked tubes, the pressure drop is usually negligible, and for an ideal gas, \( \alpha T = 1 \). For both of these cases, we have

**Ideal gas, or neglect pressure drop.**

\[
Q \rho \hat{C}_P \frac{dT}{dV} = - \sum_i \Delta H_{ri} r_i + \dot{q}
\]  

Equation 6.51 is the usual energy balance for PFRs in this chapter. The next chapter considers packed-bed reactors in which the pressure drop may be significant.
Example 6.4: PFR and interstage cooling

Consider the reversible, gas-phase reaction

$$A \xrightarrow{k_1 \rightarrow k_{-1}} B$$

The reaction is carried out in two long, adiabatic, plug-flow reactors with an interstage cooler between them as shown below $\dot{Q}$

- The feed consists of component A diluted in an inert N$_2$ stream, $N_{Af}/N_{If} = 0.1$, $N_{Bf} = 0$, and $Q_f = 10,000$ ft$^3$/hr at $P_f = 180$ psia and $T_f = 830^\circ$R.
Because the inert stream is present in such excess, we assume that the heat capacity of the mixture is equal to the heat capacity of nitrogen and is independent of temperature for the temperature range we expect.

The heat of reaction is $\Delta H_R = -5850$ BTU/lbmol and can be assumed constant. The value of the equilibrium constant is $K = k_1/k_{-1} = 1.5$ at the feed temperature.
Questions I

1. Write down the mole and energy balances that would apply in the reactors. Make sure all variables in your equations are expressed in terms of $T$ and $N_A$. What other assumptions did you make?

2. If the reactors are long, we may assume that the mixture is close to equilibrium at the exit. Using the mole balance, express $N_A$ at the exit of the first reactor in terms of the feed conditions and the equilibrium constant, $K$.

3. Using the energy balance, express $T$ at the exit of the first reactor in terms of the feed conditions and $N_A$.

4. Notice we have two equations and two unknowns because $K$ is a strong function of $T$. Solve these two equations numerically and determine the temperature and conversion at the exit of the first reactor. Alternatively, you can substitute the material balance into the energy balance to obtain one equation for $T$. Solve this equation to determine the temperature at the exit of the first reactor. What is the conversion at the exit of the first reactor?

5. Assume that economics dictate that we must run this reaction to 70% conversion to make a profit. How much heat must be removed in the interstage cooler to be able to achieve this conversion at the exit of the second reactor? What are the temperatures at the inlet and outlet of the second reactor?

6. How would you calculate the actual conversion achieved for two PFRs of specified sizes (rather than “long” ones) with this value of $\dot{Q}$?
The steady-state molar flow of A is given by the PFR material balance

\[
\frac{dN_A}{dV} = R_A = -r
\]  

(6.53)

and the rate expression for the reversible reaction is given by

\[r = k_1 c_A - k_{-1} c_B = (k_1 N_A - k_{-1} N_B)/Q\]

The molar flow of B is given by \(dN_B/dV = r\), so we conclude

\[N_B = N_{Af} + N_{Bf} - N_A = N_{Af} - N_A\]

If we assume the mixture behaves as an ideal gas at these conditions, \(c = P/RT\) or

\[Q = \frac{RT}{P} \sum_{j=1}^{n_s} N_j\]

The material balance for inert gives \(dN_i/dV = 0\), so we have the total molar flow is

\[\sum_{j=1}^{n_s} N_j = N_{Af} + N_{If}\]

and the volumetric flowrate is

\[Q = \frac{RT}{P} (N_{Af} + N_{If})\]
and the reaction rate is

\[ r = \frac{P}{RT} \left( k_1 N_A - k_{-1} (N_{Af} - N_A) \right) \left( \frac{N_{Af}}{N_{Af} + N_{If}} \right) \]

which is in terms of \( T \) and \( N_A \). The adiabatic PFR energy balance for an ideal gas is given by

\[ \frac{dT}{dV} = -\frac{\Delta H_R}{Q \rho \hat{C}_p} r \]  

(6.54)

For long reactors, \( r = 0 \) or

\[ k_1 N_A - k_{-1} (N_{Af} - N_A) = 0 \]

Dividing by \( k_{-1} \) and solving for \( N_A \) gives

\[ N_A = \frac{N_{Af}}{1 + K_1} \]
Substituting \( r = -\frac{dN_A}{dV} \) into the energy balance and multiplying through by \( dV \) gives

\[
dT = \frac{\Delta H_R}{Q\rho \hat{C}_P} dN_A
\]

The term \( Q\rho = m \) in the denominator is the mass flowrate, which is constant and equal to the feed mass flowrate. If we assume the heat of reaction and the heat capacity are weak functions of temperature and composition, we can perform the integral yielding

\[
T_1 - T_{1f} = \frac{\Delta H_R}{m \hat{C}_P} (N_A - N_{Af})
\]

\[
T = 830 + 80.1 \left( \frac{1}{1 + 0.0432 e^{2944/T} - 1} \right) = 0,
\]

\[
T_1 = 874^\circ R, \quad x = 0.56
\]

\( \dot{Q} = 200,000 \text{ BTU/hr}, \quad T_{2f} = 726^\circ R, \quad T_2 = 738^\circ R \)

Integrate Equations 6.53 and 6.54.

The results are summarized in Figure 6.34.
Figure 6.34: Temperatures and molar flows for tubular reactors with interstage cooling.
For exothermic, gas-phase reactions in a PFR, the heat release generally leads to the formation of a reactor hot spot, a point along the reactor length at which the temperature profile achieves a maximum.

If the reaction is highly exothermic, the temperature profile can be very sensitive to parameters, and a small increase in the inlet temperature or reactant feed concentration, for example, can lead to large changes in the temperature profile.

A sudden, large increase in the reactor temperature due to a small change in feed conditions is known as reactor runaway.

Reactor runaway is highly dangerous, and operating conditions are normally chosen to keep reactors far from the runaway condition.
Example 6.5: Oxidation of o-xylene to phthalic anhydride

The gas-phase oxidation of o-xylene to phthalic anhydride

\[
\begin{align*}
\text{C}_8\text{H}_8 + 3\text{O}_2 & \rightarrow \text{C}_8\text{H}_8\text{O}_4 + 3\text{H}_2\text{O} \\
\text{CH}_3 & \text{C} \text{CH}_3 \\
& \quad + \quad 3 \text{O}_2 \\
& \quad \quad \rightarrow \\
& \quad \quad \quad \text{C}_8\text{H}_8\text{O}_4 \\
& \quad \quad \quad \quad + \quad 3 \text{H}_2\text{O}
\end{align*}
\]

is highly exothermic.

The reaction is carried out in PFR tube bundles with molten salt circulating as the heat transfer fluid [2]. The o-xylene is mixed with air before entering the PFR. The reaction rate is limited by maintaining a low concentration of hydrocarbon in the feed. The mole fraction of o-xylene is less than 2%.

Under these conditions, the large excess of oxygen leads to a pseudo-first-order rate expression

\[
r = k_m \exp \left[ -E \left( \frac{1}{T} - \frac{1}{T_m} \right) \right] c_x
\]
in which $c_x$ is the o-xylene concentration. The operating pressure is atmospheric. Calculate the temperature and o-xylene composition profiles. The kinetic parameters are adapted from Van Welsenaere and Froment and given in Table 6.5 [7].

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Value</th>
<th>Units</th>
</tr>
</thead>
<tbody>
<tr>
<td>$k_m$</td>
<td>1922.6</td>
<td>s$^{-1}$</td>
</tr>
<tr>
<td>$T_a$</td>
<td>625</td>
<td>K</td>
</tr>
<tr>
<td>$T_m$</td>
<td>625</td>
<td>K</td>
</tr>
<tr>
<td>$P_f$</td>
<td>1.0</td>
<td>atm</td>
</tr>
<tr>
<td>$l$</td>
<td>1.5</td>
<td>m</td>
</tr>
<tr>
<td>$R$</td>
<td>0.0125</td>
<td>m</td>
</tr>
<tr>
<td>$\hat{C}_p$</td>
<td>0.992</td>
<td>kJ/kg K</td>
</tr>
<tr>
<td>$U^o$</td>
<td>0.373</td>
<td>kJ/m$^2$ s K</td>
</tr>
<tr>
<td>$y_{xf}$</td>
<td>0.019</td>
<td></td>
</tr>
<tr>
<td>$E/R$</td>
<td>$1.3636 \times 10^4$</td>
<td>K</td>
</tr>
<tr>
<td>$\Delta H_R$</td>
<td>$-1.361 \times 10^3$</td>
<td>kJ/kmol</td>
</tr>
<tr>
<td>$Q_\rho$</td>
<td>$2.6371 \times 10^{-3}$</td>
<td>kg/s</td>
</tr>
</tbody>
</table>

Table 6.5: PFR operating conditions and parameters for o-xylene example.
If we assume constant thermochemical properties, an ideal gas mixture, and express the mole and energy balances in terms of reactor length, we obtain

\[
\frac{dN_x}{dz} = -A_c r \\
\frac{dT}{dz} = -\beta r + \gamma (T_a - T) \\
r = \frac{kP}{RT} \frac{N_x}{N}
\]

in which

\[
\beta = \frac{\Delta H_R A_c}{Q \rho \hat{C}_P}, \quad \gamma = \frac{2\pi RU^\circ}{Q \rho \hat{C}_P}
\]

and the total molar flow is constant and equal to the feed molar flow because of the stoichiometry.

Figure 6.35 shows the molar flow of o-xylene versus reactor length for several values of the feed temperature. The corresponding temperature profile is shown in Figure 6.36.
Figure 6.35: Molar flow of o-xylene versus reactor length for different feed temperatures.
**Figure 6.36**: Reactor temperature versus length for different feed temperatures.
We see a hotspot in the reactor for each feed temperature.

Notice the hotspot temperature increases and moves down the tube as we increase the feed temperature.

Finally, notice if we increase the feed temperature above about 631 K, the temperature spikes quickly to a large value and all of the o-xylene is converted by $z = 0.6 \text{ m}$, which is a classic example of reactor runaway.

To avoid this reactor runaway, we must maintain the feed temperature below a safe value.

This safe value obviously also depends on how well we can control the composition and temperature in the feed stream. Tighter control allows us to operate safely at higher feed temperatures and feed o-xylene mole fractions, which increases the production rate.
In many applications, it is necessary to heat a feed stream to achieve a reactor inlet temperature having a high reaction rate. If the reaction also is exothermic, we have the possibility to lower the reactor operating cost by heat integration. The essential idea is to use the heat released by the reaction to heat the feed stream. As a simple example of this concept, consider the following heat integration scheme.
Figure 6.37: Autothermal plug-flow reactor; the heat released by the exothermic reaction is used to preheat the feed.
This reactor configuration is known as an autothermal plug-flow reactor. The reactor system is an annular tube. The feed passes through the outer region and is heated through contact with the hot reactor wall. The feed then enters the inner reaction region, which is filled with the catalyst, and flows countercurrently to the feed stream. The heat released due to reaction in the inner region is used to heat the feed in the outer region. When the reactor is operating at steady state, no external heat is required to preheat the feed. Of course, during the reactor start up, external heat must be supplied to ignite the reactor.
Although recycle of energy can offer greatly lower operating costs, the dynamics and control of these reactors may be complex. We next examine an ammonia synthesis example to show that multiple steady states are possible. Ammonia synthesis had a large impact on the early development of the chemical engineering discipline. Quoting Aftalion [1, p. 101]
While physicists and chemists were linking up to understand the structure of matter and giving birth to physical chemistry, another discipline was emerging, particularly in the United States, at the beginning of the twentieth century, that of chemical engineering... it was undoubtedly the synthesis of ammonia by BASF, successfully achieved in 1913 in Oppau, which forged the linking of chemistry with physics and engineering as it required knowledge in areas of analysis, equilibrium reactions, high pressures, catalysis, resistance of materials, and design of large-scale apparatus.
Example 6.6: Ammonia synthesis

Calculate the steady-state conversion for the synthesis of ammonia using the autothermal process shown previously.

A rate expression for the reaction

\[
\text{N}_2 + 3\text{H}_2 \xrightleftharpoons{k_1}{k_{-1}} 2\text{NH}_3
\]

over an iron catalyst at 300 atm pressure is suggested by Temkin [4]

\[
r = \frac{k_{-1}}{RT} \left[ K^2(T) \frac{P_N P_H^{3/2}}{P_A} - \frac{P_A}{P_H^{3/2}} \right]
\]

in which \(P_N\), \(P_H\), \(P_A\) are the partial pressures of nitrogen, hydrogen, and ammonia, respectively, and \(K\) is the equilibrium constant for the reaction forming one mole of ammonia.

For illustration, we assume the thermochemical properties are constant and the gases form an ideal-gas mixture.

More accurate thermochemical properties and a more accurate equation of state do not affect the fundamental behavior predicted by the reactor model.
The steady-state material balance for the ammonia is

\[ \frac{dN_A}{dV} = R_A = 2r \]

\[ N_A(0) = N_{Af} \]

and the other molar flows are calculated from

\[ N_N = N_{Nf} - \frac{1}{2}(N_A - N_{Af}) \]

\[ N_H = N_{Hf} - \frac{3}{2}(N_A - N_{Af}) \]

If we assume an ideal gas in this temperature and pressure range, the volumetric flowrate is given by

\[ Q = \frac{RT}{P}(N_A + N_N + N_H) \]

The energy balance for the reactor is the usual

\[ Q \rho \hat{C}_p \frac{dT}{dV} = -\Delta H_R r + \dot{q} \quad (6.55) \]

in which \( \dot{q} \) is the heat transfer taking place between the reacting fluid and the cold feed

\[ \dot{q} = \frac{2}{R} U^o (T_a - T) \]
### Table 6.6: Parameter values for ammonia example

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Value</th>
<th>Units</th>
</tr>
</thead>
<tbody>
<tr>
<td>$P$</td>
<td>300</td>
<td>atm</td>
</tr>
<tr>
<td>$Q_0$</td>
<td>0.16</td>
<td>m$^3$/s</td>
</tr>
<tr>
<td>$A_c$</td>
<td>1</td>
<td>m$^2$</td>
</tr>
<tr>
<td>$l$</td>
<td>12</td>
<td>m</td>
</tr>
<tr>
<td>$T_{af}$</td>
<td>323</td>
<td>K</td>
</tr>
<tr>
<td>$\gamma$</td>
<td>$\frac{2\pi R U^o}{Q \rho \hat{C}_P}$</td>
<td>0.5</td>
</tr>
<tr>
<td>$\beta$</td>
<td>$\frac{\Delta H_R A_c}{Q \rho \hat{C}_P}$</td>
<td>$-2.342$</td>
</tr>
<tr>
<td>$\Delta G^o$</td>
<td>4250</td>
<td>cal/mol</td>
</tr>
<tr>
<td>$\Delta H^o$</td>
<td>$-1.2 \times 10^4$</td>
<td>cal/mol</td>
</tr>
<tr>
<td>$k_{-10}$</td>
<td>$7.794 \times 10^{11}$</td>
<td>cal/mol</td>
</tr>
<tr>
<td>$E_{-1}/R$</td>
<td>$2 \times 10^4$</td>
<td>K</td>
</tr>
</tbody>
</table>

"Ammonia synthesis III"
The material balances for the feed-heating section are simple because reaction does not take place without the catalyst. Without reaction, the molar flow of all species are constant and equal to their feed values and the energy balance for the feed-heating section is

$$Q_a \rho_a \dot{C}_{Pa} \frac{dT_a}{dV_a} = -\dot{q}$$

(6.56)

$$T_a(0) = T_{af}$$

in which the subscript $a$ represents the fluid in the feed-heating section. Notice the heat terms are of opposite signs in Equations 6.56 and 6.55. If we assume the fluid properties do not change significantly over the temperature range of interest, and switch the direction of integration in Equation 6.56 using $dV_a = -dV$, we obtain

$$Q \rho \dot{C}_P \frac{dT_a}{dV} = \dot{q}$$

$$T_a(\text{V}_R) = T_{af}$$

(6.58)

Finally we require a boundary condition for the reactor energy balance, which we have from the fact that the heating fluid enters the reactor at $z = 0$, $T(0) = T_a(0)$. 
Combining these balances and boundary conditions and converting to reactor length in place of volume gives the model

\[
\begin{align*}
\frac{dN_A}{dz} &= 2A_c r \\
\frac{dT}{dz} &= -\beta r + \gamma(T_a - T) \\
\frac{dT_a}{dz} &= \gamma(T_a - T)
\end{align*}
\]

\(N_A(0) = N_{Af}\)
\(T(0) = T_a(0)\)
\(T_a(l) = T_{af}\) \(\tag{6.59}\)

in which

\[\beta = \frac{\Delta H_R A_c}{Q \rho \hat{C}_P}, \quad \gamma = \frac{2\pi RU^o}{Q \rho \hat{C}_P}\]

Equation 6.59 is a boundary-value problem, rather than an initial-value problem, because \(T_a\) is specified at the exit of the reactor. A simple solution strategy is to guess the reactor inlet temperature, solve the model to the exit of the reactor, and then compare the computed feed preheat temperature to the specified value \(T_{af}\). This strategy is known as a shooting method. We guess the missing values required to produce an initial-value problem. We solve the initial-value problem, and then iterate on the guessed values until we match the specified boundary conditions. We will see more about boundary-value problems and shooting methods when we treat diffusion in Chapter 7. \(\square\)
Figure 6.38 shows the results for the parameter values listed in Table 6.6, which are based on those used by van Heerden [6].

For given values of $T_a(0)$, we solve the initial-value problem, Equation 6.59, and plot the resulting $T_a(V_R)$ as the solid line in Figure 6.38.

The intersection of that line with the feed temperature $T_{af} = 323 \text{ K}$ indicates a steady-state solution.

Notice three steady-state solutions are indicated in Figure 6.38 for these values of parameters.
Solution III

Figure 6.38: Coolant temperature at reactor outlet versus temperature at reactor inlet, $T_a(l)$ versus $T_a(0)$; intersection with coolant feed temperature $T_{af}$ indicates three steady-state solutions (A,B,C).

The profiles in the reactor for these three steady states are shown in Figures 6.39 and 6.40. It is important to operate at the upper steady state so that a reasonably large production of ammonia is achieved.
Figure 6.39: Reactor and coolant temperature profiles versus reactor length; lower (A), unstable middle (B), and upper (C) steady states.
Figure 6.40: Ammonia mole fraction versus reactor length; lower (A), unstable middle (B), and upper (C) steady states.
Tables 6.7–6.10 summarize the important energy balances for the batch, continuous-stirred-tank, semi-batch, and plug-flow reactors.

In contrast to the material balance, which is reasonably straightforward, choosing the proper energy balance requires some care. It is unwise to select an energy balance from a book without carefully considering the assumptions that have been made in the derivation of that particular energy balance.

Hopefully these tables will help you to choose an appropriate energy balance.
Neglect kinetic and potential energies

\[
\frac{dU}{dt} = \dot{Q} + \dot{W}_s + \dot{W}_b
\]

Neglect shaft work

\[
\frac{dU}{dt} + P \frac{dV_R}{dt} = \dot{Q}
\]

\[
\frac{dH}{dt} - V_R \frac{dP}{dt} = \dot{Q}
\]

Single phase

\[
V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha TV_R \frac{dP}{dt} + \sum_j H_j \frac{dn_j}{dt} = \dot{Q}
\]

\[
V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha TV_R \frac{dP}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \dot{Q}
\]

a. Incompressible-fluid or constant-pressure reactor

\[
V_R \rho \hat{C}_P \frac{dT}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \dot{Q}
\]
### b. Constant-volume reactor

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i \left( \Delta H_{R_i} - \frac{\alpha}{\kappa T} T \Delta V_{R_i} \right) r_i V_R + \dot{Q} \]

### b.1 Constant-volume reactor, ideal gas

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = - \sum_i \left( \Delta H_{R_i} - RT \bar{\nu}_i \right) r_i V_R + \dot{Q} \]

**Table 6.7:** Energy balances for the batch reactor.
Neglect kinetic and potential energies

\[
\frac{dU}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q} + \dot{W}_s + \dot{W}_b
\]

Neglect shaft work

\[
\frac{dU}{dt} + P \frac{dV_R}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q}
\]

\[
\frac{dH}{dt} - V_R \frac{dP}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q}
\]

Single phase

\[
V_R \rho \hat{C}_p \frac{dT}{dt} - \alpha T V_R \frac{dP}{dt} + \sum_j \overline{H}_j \frac{dn_j}{dt} = Q_f \rho_f \hat{H}_f - Q \rho \hat{H} + \dot{Q}
\]

\[
V_R \rho \hat{C}_p \frac{dT}{dt} - \alpha T V_R \frac{dP}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q}
\]

a. Incompressible-fluid or constant-pressure reactor

\[
V_R \rho \hat{C}_p \frac{dT}{dt} = - \sum_i \Delta H_{Ri} r_i V_R + \sum_j c_{jf} Q_f (\overline{H}_{jf} - \overline{H}_j) + \dot{Q}
\]
b. Constant-volume reactor

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = -\sum_i (\Delta H_{Ri} - \frac{\alpha}{\kappa T} T \Delta V_{Ri}) r_i V_R + \sum_j c_{jf} Q_f (\bar{H}_{jf} - \bar{H}_j) + \frac{\alpha}{\kappa T} T \sum_j V_j (c_{jf} Q_f - c_j Q) + \dot{Q} \]

b.1 Constant-volume reactor, ideal gas

\[ V_R \rho \hat{C}_V \frac{dT}{dt} = -\sum_i (\Delta H_{Ri} - RT \bar{v}_i) r_i V_R + \sum_j c_{jf} Q_f (\bar{H}_{jf} - \bar{H}_j) + RT \sum_j (c_{jf} Q_f - c_j Q) + \dot{Q} \]

c. Steady state, constant \( \hat{C}_P, P = P_f \)

\[ -\sum_i \Delta H_{Ri} r_i V_R + Q_f \rho_f \hat{C}_P (T_f - T) + \dot{Q} = 0 \]

Table 6.8: Energy balances for the CSTR.
Neglect kinetic and potential energies

\[ \frac{dU}{dt} = Q_f \rho_f \hat{H}_f + \dot{Q} + \dot{W}_s + \dot{W}_b \]

Neglect shaft work

\[ \frac{dU}{dt} + P \frac{dV}{dt} = \dot{Q}_f \rho_f \hat{H}_f + \dot{Q} \]
\[ \frac{dH}{dt} - V_R \frac{dP}{dt} = \dot{Q}_f \rho_f \hat{H}_f + \dot{Q} \]

Single phase

\[ V_R \rho \hat{C}_P \frac{dT}{dt} - \alpha T V_R \frac{dP}{dt} + \sum_j \bar{H}_j \frac{dn_j}{dt} = \dot{Q}_f \rho_f \hat{H}_f + \dot{Q} \]
\[ V_R \rho \hat{C}_P \frac{dT}{dt} = \sum_i \Delta H_{R_i} r_i V_R + \sum_j c_{jf} Q_f (\bar{H}_{jf} - \bar{H}_j) + \dot{Q} \]

a. Incompressible-fluid or constant-pressure reactor

\[ V_R \rho \hat{C}_P \frac{dT}{dt} = \sum_i \Delta H_{R_i} r_i V_R + \sum_j c_{jf} Q_f (\bar{H}_{jf} - \bar{H}_j) + \dot{Q} \]

a.1 Constant \( \hat{C}_P \)

\[ V_R \rho \hat{C}_P \frac{dT}{dt} = \sum_i \Delta H_{R_i} r_i V_R + Q_f \rho_f \hat{C}_P (T_f - T) + \dot{Q} \]

Table 6.9: Energy balances for the semi-batch reactor.
Neglect kinetic and potential energies and shaft work

\[
\frac{\partial}{\partial t} (\rho \hat{U}) = - \frac{1}{A_c} \frac{\partial}{\partial z} (Q \rho \hat{H}) + \dot{q}
\]

Heat transfer with an overall heat-transfer coefficient

\[
\dot{q} = \frac{2}{R} U^o (T_a - T)
\]

Steady state

\[
\frac{d}{dV} (Q \rho \hat{H}) = \dot{q}
\]

Single phase

\[
Q \rho \hat{C}_p \frac{dT}{dV} + Q (1 - \alpha T) \frac{dP}{dV} = - \sum_i \Delta H_{Ri} r_i + \dot{q}
\]

a. Neglect pressure drop, or ideal gas

\[
Q \rho \hat{C}_p \frac{dT}{dV} = - \sum_i \Delta H_{Ri} r_i + \dot{q}
\]

b. Incompressible fluid

\[
Q \rho \hat{C}_p \frac{dT}{dV} + Q \frac{dP}{dV} = - \sum_i \Delta H_{Ri} r_i + \dot{q}
\]

Table 6.10: Energy balances for the plug-flow reactor.
Nonisothermal reactor design requires the simultaneous solution of the appropriate energy balance and the species material balances. For the batch, semi-batch, and steady-state plug-flow reactors, these balances are sets of initial-value ODEs that must be solved numerically.

In very limited situations (constant thermodynamic properties, single reaction, adiabatic), one can solve the energy balance to get an algebraic relation between temperature and concentration or molar flowrate.

The nonlinear nature of the energy and material balances can lead to multiple steady-state solutions. Steady-state solutions may be unstable, and the reactor can exhibit sustained oscillations. These reactor behaviors were illustrated with exothermic CSTRs and autothermal tubular reactors.
Notation I

- $A$: heat transfer area
- $A_c$: reactor tube cross-sectional area
- $A_i$: preexponential factor for rate constant $i$
- $c_j$: concentration of species $j$
- $c_{jf}$: feed concentration of species $j$
- $c_{js}$: steady-state concentration of species $j$
- $c_{j0}$: initial concentration of species $j$
- $C_P$: constant-pressure heat capacity
- $\bar{C}_P$: partial molar heat capacity
- $C_{Ps}$: heat capacity per volume
- $\hat{C}_P$: constant-pressure heat capacity per mass
- $\hat{C}_V$: constant-volume heat capacity per mass
- $\Delta C_P$: heat capacity change on reaction, $\Delta C_P = \sum_j \nu_j \bar{C}_P j$
- $E_i$: activation energy for rate constant $i$
- $E_k$: total energy of stream $k$
- $\hat{E}_k$: total energy per mass of stream $k$
- $\Delta G^\circ$: Gibbs energy change on reaction at standard conditions
- $\bar{H}_j$: partial molar enthalpy
- $\hat{H}$: enthalpy per unit mass
- $\Delta H_{Ri}$: enthalpy change on reaction, $\Delta H_{Ri} = \sum_j \nu_{ij} \bar{H}_j$
- $\Delta H^\circ$: enthalpy change on reaction at standard conditions
- $k_i$: reaction rate constant for reaction $i$
- $k_m$: reaction rate constant evaluated at mean temperature $T_m$
- $K_i$: equilibrium constant for reaction $i$
- $\hat{K}$: kinetic energy per unit mass
- $l$: tubular reactor length
- $m_k$: total mass flow of stream $k$
Notation II

\( n \)  
reaction order

\( n_j \)  
moles of species \( j \), \( V_R c_j \)

\( n_r \)  
number of reactions in the reaction network

\( n_s \)  
number of species in the reaction network

\( N_j \)  
molar flow of species \( j \), \( Qc_j \)

\( N_{j\text{f}} \)  
feed molar flow of species \( j \), \( Qc_j \)

\( P \)  
pressure

\( P_j \)  
partial pressure of component \( j \)

\( \dot{q} \)  
heat transfer rate per volume for tubular reactor, \( \dot{q} = \frac{2}{R} U^O (T_a - T) \)

\( Q \)  
volumetric flowrate

\( Q_f \)  
feed volumetric flowrate

\( \dot{Q} \)  
heat transfer rate to reactor, usually modeled as \( \dot{Q} = U^O A (T_a - T) \)

\( r_i \)  
reaction rate for \( i \)th reaction

\( r_{\text{tot}} \)  
total reaction rate, \( \sum_i r_i \)

\( R \)  
gas constant

\( R_j \)  
production rate for \( j \)th species

\( t \)  
time

\( T \)  
temperature

\( T_a \)  
temperature of heat transfer medium

\( T_m \)  
mean temperature at which \( k \) is evaluated

\( U^O \)  
overall heat transfer coefficient

\( \hat{U} \)  
internal energy per mass

\( v_k \)  
velocity of stream \( k \)

\( V \)  
reactor volume variable

\( V_j \)  
partial molar volume of species \( j \)

\( V_j^O \)  
specific molar volume of species \( j \)

\( V_R \)  
reactor volume
Notation III

$\Delta V_i$ change in volume upon reaction $i$, $\sum_j \nu_{ij} \bar{V}_j$

$\dot{W}$ rate work is done on the system

$x_j$ number of molecules of species $j$ in a stochastic simulation

$x_j$ molar conversion of species $j$

$y_j$ mole fraction of gas-phase species $j$

$z$ reactor length variable

$\alpha$ coefficient of expansion of the mixture, $\alpha = (1/V)(\partial V/\partial T)_{P, n_j}$

$\varepsilon_i$ extent of reaction $i$

$\tau$ reactor residence time, $\tau = V_R/Q_f$

$\nu_{ij}$ stoichiometric coefficient for species $j$ in reaction $i$

$\bar{\nu}_i$ $\sum_j \nu_{ij}$

$\rho$ mass density

$\rho_k$ mass density of stream $k$

$\hat{\Phi}$ potential energy per mass
Thermodynamics Summary

1. \( \alpha := \frac{1}{V} \left( \frac{\partial V}{\partial T} \right)_{P,n_j} \)  
   coefficient of expansion

2. \( \kappa_T := -\frac{1}{V} \left( \frac{\partial V}{\partial P} \right)_{T,n_j} \)  
   isothermal compressibility

3. \( H := U + PV \)  
   enthalpy

4. \( H_j := \left( \frac{\partial H}{\partial n_j} \right)_{T,P,n_k} \)  
   partial molar enthalpy (property)

5. \( \left( \frac{\partial H}{\partial T} \right)_{P,n_j} := C_P := \rho V \hat{C}_P; \quad \left( \frac{\partial U}{\partial T} \right)_{V,n_j} := C_V := \rho V \hat{C}_V \)  
   heat capacities

6. \( \left( \frac{\partial H}{\partial P} \right)_{T,n_j} = V (1 - \alpha T) \)

7. \( \left( \frac{\partial P}{\partial T} \right)_{V,n_j} = \frac{\alpha}{\kappa_T} \)

8. \( C_P = C_V + \alpha TV \left( \frac{\partial P}{\partial T} \right)_{V,n_j} = C_V + \frac{\alpha^2}{\kappa_T} TV \)

9. \( \left( \frac{\partial P}{\partial n_j} \right)_{T,V,n_k} = \frac{\bar{V}_j}{\bar{V} \kappa_T} \)

10. Incompressible fluid: \( \alpha = 0, \hat{C}_V = \hat{C}_P \)

11. Ideal gas: \( PV = nRT, \alpha T = 1, \kappa_T P = 1, C_P = C_V + nR, \bar{V}_j = \frac{RT}{P}, \Delta V_{Ri} = \bar{v}_i \frac{RT}{P} \)
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